CE 809 - Structural Dynamics

Lecture 5: Response of SDF Systems to Impulse Loading

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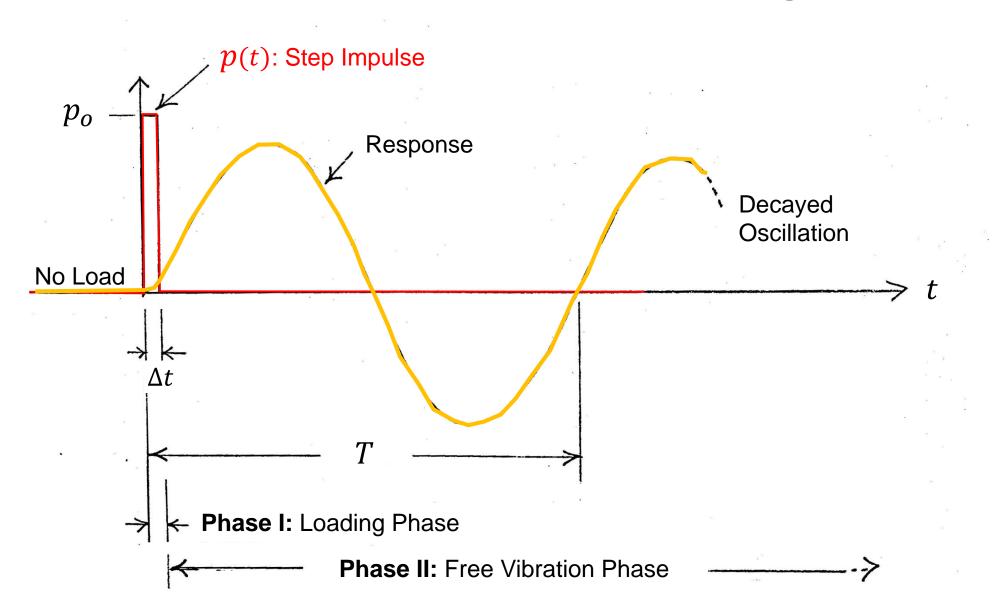
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Impulse Loading

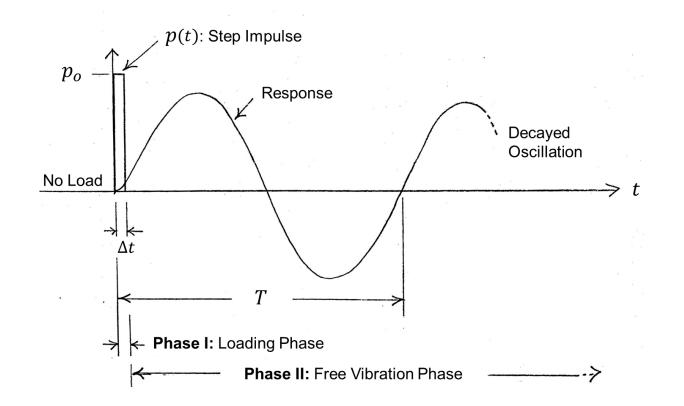
- Impulsive forces, shock loads, short duration loads.
- Impact, blast wave, explosion
- Sudden stop/break of trucks & automobiles & travelling cranes

The study of impulse response is also important for the **analysis of response to arbitrary loadings**.

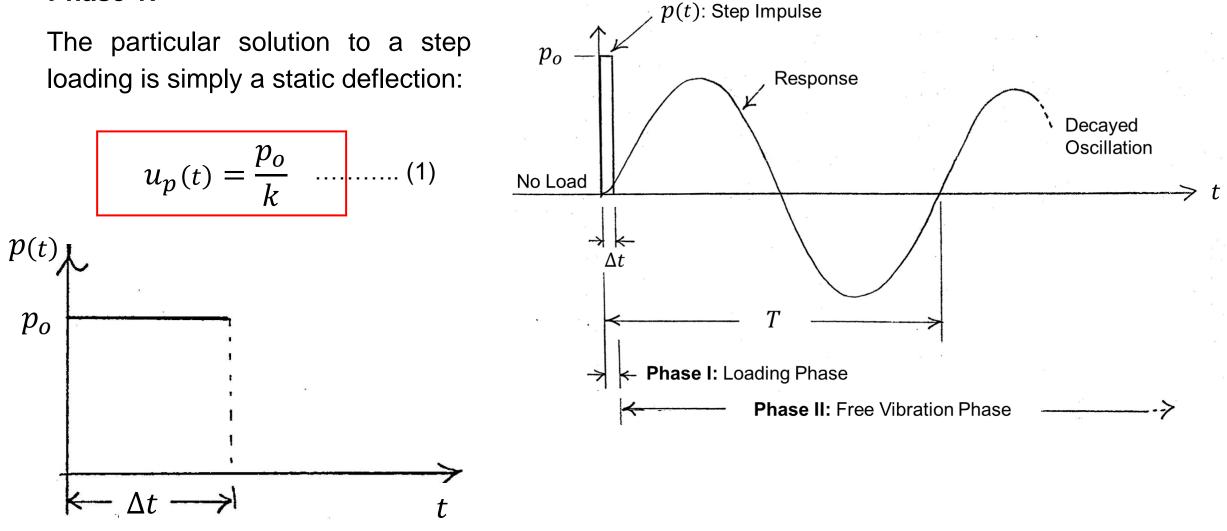


Impulse Force Magnitude = p_o , start at t = 0, Duration = Δt , where $\Delta t/T << 1$

Structure Initial at-rest: u(0) = 0, $\dot{u}(0) = 0$



Phase 1:



Phase 1:

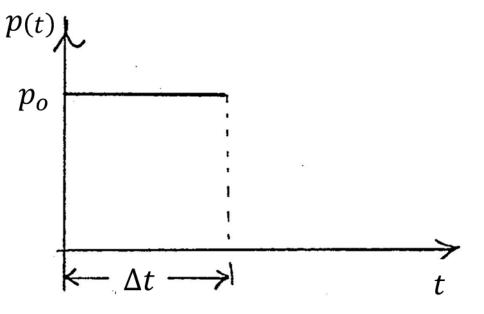
$$u_p(t) = \frac{p_o}{k}$$

This solution satisfies the equation of motion:

$$\dot{u}_p(t) = 0$$
$$\ddot{u}_p(t) = 0$$

Substitute the above relations into the equation of motion:

$$m \ddot{u}_{p}(t) + c \dot{u}_{p}(t) + k u_{p}(t) = p_{0}$$
$$0 + 0 + k \frac{p_{0}}{k} = p_{0}$$



Phase 1:

A general solution:

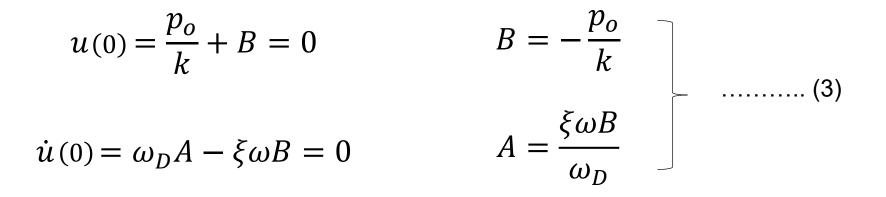
$$u(t) = u_p(t) + u_h(t)$$

$$u(t) = \frac{p_o}{k} + e^{-\xi \,\omega t} \left[A \sin(\omega_D t) + B \cos(\omega_D t) \right] \qquad \dots \dots \dots \dots (2)$$

Where

A and B are determined such that the at-rest initial conditions are satisfied.

Phase 1:



So we obtain:

$$u(t) = \frac{p_o}{k} + e^{-\xi \,\omega t} \left[-\frac{\xi \,\omega}{\omega_D} \,\frac{p_o}{k} \sin(\omega_D t) - \frac{p_o}{k} \cos(\omega_D t) \right] \qquad \dots \dots (4)$$

(valid in the range $0 < t < \Delta t$)

Phase 1:

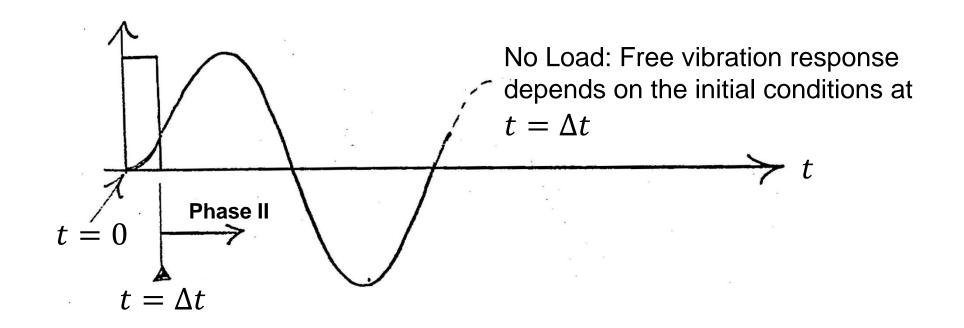
$$u(t) = \frac{p_o}{k} + e^{-\xi \,\omega t} \left[-\frac{\xi \,\omega}{\omega_D} \,\frac{p_o}{k} \sin(\omega_D t) - \frac{p_o}{k} \cos(\omega_D t) \right] \qquad \dots \dots (4)$$

Due to fact that $0 < t < \Delta t$ and $\Delta t/T << 1$ and $\xi << 1$

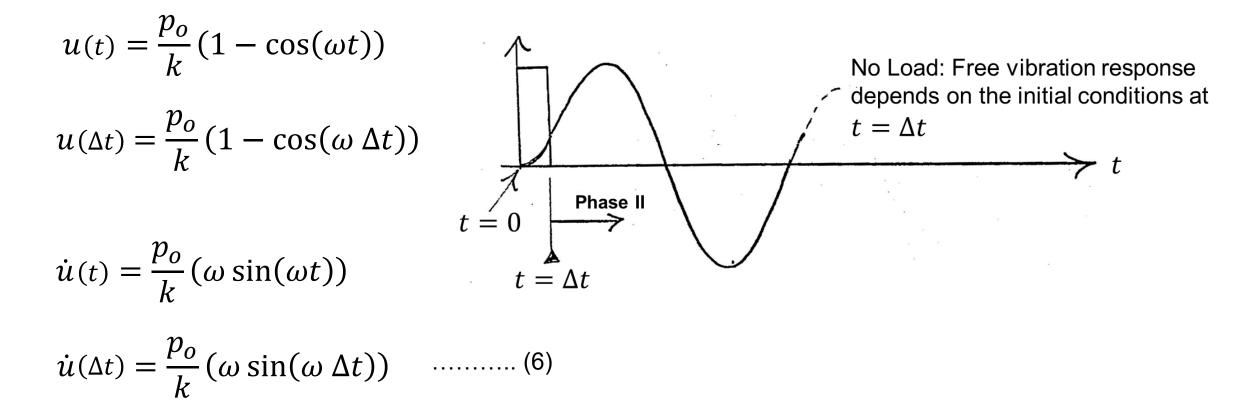
$$u(t) \approx \frac{p_o}{k} - \frac{p_o}{k} \cos(\omega t) \approx \frac{p_o}{k} (1 - \cos(\omega t)) \qquad \dots \dots \dots \dots (5)$$

Equation (5) shows the approximate response during the loading phase.

Phase 2:



Phase 2:



Phase 2:

Employing Tylor's expansion:

$$\sin \theta = \theta - \frac{\theta^3}{3!} + \frac{\theta^5}{5!} \dots \dots$$
$$\cos \theta = 1 - \frac{\theta^2}{2!} + \frac{\theta^4}{4!} \dots \dots$$

For small θ , by neglecting the second order term and higher terms, we get

$$\sin\theta \approx \theta$$
, $\cos\theta \approx 1$ (7)

Phase 2:

Introducing this approximation in Equation (7) into Equation (6), we obtain

$$u(\Delta t) = \frac{p_o}{k} (1 - \cos(\omega \Delta t)) \cong 0$$

$$\dot{u}(\Delta t) = \frac{p_o}{k} (\omega \sin(\omega \Delta t)) \cong \frac{p_o \omega^2 \Delta t}{k} = \frac{p_o \Delta t}{m}$$
(8)

Note that $p_o \Delta t$ is an impulse.

Equation (8) says that **impulse** ≈ the change in momentum (of the mass)

The impulse introduces "momentum" into a structure but the duration of impulse is so short that the displacement has not been developed yet.

Phase 2 $(t > \Delta t)$:

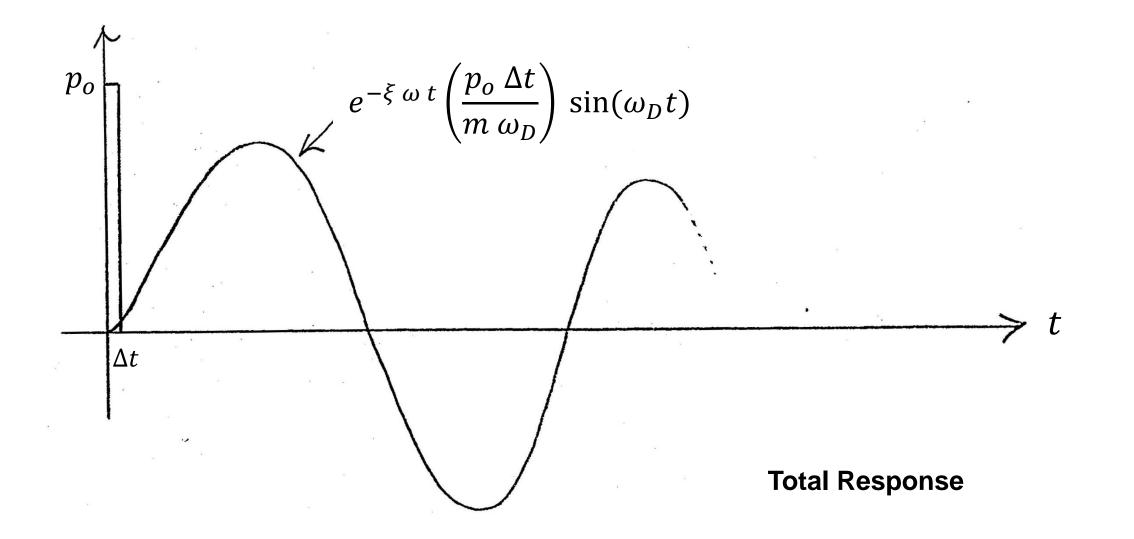
Using the Equation (8) as the initial conditions for free vibration in Phase 2,

$$u(t) = e^{-\xi \,\omega \,(t - \Delta t)} \left[\frac{\dot{u}(\Delta t)}{\omega_D} \sin(\omega_D (t - \Delta t)) \right] \quad \dots \dots (9)$$

Since $\Delta t/t << 1$, it is justified to let $t - \Delta t \approx t$, that is

$$u(t) \approx e^{-\xi \omega t} \left(\frac{p_o \Delta t}{m \omega_D} \right) \sin(\omega_D t)$$
(10)

Equation (10) will be used when we analyze the response to arbitrary loading in the next section.



Thank you